

Process Equipment Design
Prof. Shabina Khanam
Department of Chemical Engineering
Indian Institute of Technology – Roorkee

Lecture –21
STE Design – Bell's Method Example-5

Hello everyone. Welcome to the 5th week of the course Process Equipment Design and here we are with lecture 21 and this is the first lecture of 5th week and here we are going to discuss design of shell and tube heat exchanger using Bell's method. In 4th week, we have discussed design of shell and tube heat exchanger using Bell's method and different steps involved in that and further we have seen one example in 5th lecture of last week.

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Design of Shell and Tube Heat Exchanger

Example – 2

Calculate heat transfer coefficient of shell side of shell and tube exchanger using Bell's method for following specifications:

Toluene (organic liquid) with 40000 kg/h flow rate is cooled from 60°C to 15°C in a heat exchanger. 80000 kg/h of water, available at 5°C, is used as a cooling media. Densities of toluene and water are 865 kg/m³ and 950 kg/m³, respectively.

$D_s = \cancel{D_o} = 531.2\text{mm}$
 $P_t = 0.02375\text{m} ; d_o = 0.019\text{m}$

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And here we are illustrating the design of exchanger using Bell's method with few more examples. So, let us start with example 2. So, in this example you have to calculate heat transfer coefficient of shell side of a shell and tube heat exchanger using Bell's method for following specifications. Here, we have to consider toluene which is basically an organic liquid with 40,000 kg per hour flow rate.

And it is cooled from 60 degree to 15 degree Celsius in a heat exchange. 80,000 kg per hour of water is used which is available at 5 degree Celsius and this we can consider as cooling media. Densities of toluene and water are given like this and here we are given shell dia as 531.2 mm and this is not OD this is basically shell dia. Pitch is given as 0.2375 meter and outer diameter of tube is given as 19 mm.

So, you see here we are given some specification about the problem and if you see this problem this is the same problem which we have discussed in 4th week and if you see the first lecture of that week where we have illustrated the design of shell and tube heat exchanger using Kern's method. So, the same example I am considering over here, but here we should apply Bell's method.

So as far as heat transfer coefficient and tube side is concerned that you have already covered in the first lecture of week 4. So, I am not going into detail of that, I am starting from Bell's method when it should be considered to shell side heat transfer coefficient. So, let us start this. So, here as you know that when we apply Bell's method the first parameter we have to obtain is h_{oc} which is the heat transfer coefficient on ideal tube bank.

And then we will add other correction factors to find out actual shell side heat transfer coefficient.

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Design of Shell and Tube Heat Exchanger

Solution

$G_s = \frac{40000}{3600} \times \frac{1}{0.0094} = 1182.03 \checkmark$

Let $L_b = \frac{D_s}{4} \checkmark$

$A_s = 0.0141 \text{ m}^2 \checkmark$

$G_s = 788.02 \text{ kg/m}^3\text{s} \checkmark$

$u_s = 0.911 \text{ m/s} \checkmark$

Between 0.3 and 1 m/s (accepted)

$\mu_{\text{toluene}} = 0.48$

$Re = \frac{G_s d_o}{\mu} = 3.119 \times 10^4 \checkmark$

$j_d = 5 \times 10^{-3} \checkmark$

$k_{\text{toluene}} = 0.0848 \text{ W/m}^2\text{ }^\circ\text{C}, C_p = 1.767 \text{ J/g}^\circ\text{C}$

$Pr = \frac{C_p \mu}{k} = 10.001 \checkmark$

$h_{oc} \left(\frac{\mu}{\mu_w} \right)^{1/4} = j_d \checkmark$

$\frac{h_c d_o}{k_f} = j_d Re Pr^{1/3} \checkmark$

$h_{oc} = 1499.55 \text{ W/m}^2\text{ }^\circ\text{C} \quad ; Re > 2000 \text{ turbulent}$

For Δ pitch,

$P_t = 0.02066 \text{ m} = 20.66 \text{ mm}$

$H_c = 132.8 \text{ mm}$

Height between baffle tips = $D_s - 2H_c$
= 265.6 mm

So, if you see when we calculate G_s we have considered 40,000 kg per hour as the mass flow rate and which is of toluene because when I compare toluene and water you can understand that water is more corrosive, water has more faulting tendency so it should be allocated to tube side. So, toluene should be considered in shell side. So, considering that we can find out G_s value which comes out as 1182.03.

And further we have to find out baffle spacing. So, here I am considering $D_s / 4$ and I can also find out cross flow area that is A_s which is basically based on baffle spacing as well as shell dia. So, A_s value you can obtain as we have discussed in previous examples in Bell's method and Kern's method. Considering these, we can obtain shell side velocity as 0.911 and that should be U_s .

So, if I compare this velocity it is basically falling within the range because it should be within 0.321 meter per second. So, whatever U_s we have obtained that we can accept and as far as viscosity of toluene is concerned because that will be used to find out Reynolds number and viscosity of toluene, water all these we have already seen in the first lecture of 4th week. So, we can simply use these values instead of collecting the properties again.

So, once you have the viscosity of toluene you can find out Reynolds number which is coming out as 3.119 into 10 is to the power 4 and corresponding to this we can find out j_h factor. So, here it should be j_h , so we can find out j_h factor depending upon the Reynolds number and further we can find out Prandtl number so it comes as 10.001 and we can ignore viscosity correction factor so that I have (j_h) (05:46).

And further you can find out h_{oc} value using this expression and value you can obtain as 1499.55 watt per meter square degree Celsius because I am having Reynolds number more than 2,000 and after that once I am having h_{oc} value after that we have to find out different correction factor like F_n F_w F_b and F_L . So to start this, we should first focus on F_n correction factor.

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Design of Shell and Tube Heat Exchanger

Solution

$$N_{CV} = \frac{D_s - 2H_c}{P_t} = 12.85$$

$$N_{CV} = 13$$

$$F_n = 1.01$$

$$H_b = \frac{D_b}{2} - D_s (0.5 - 0.25) \text{ [Let 25% Baffle cut]}$$

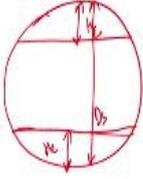
$$= 126.55 \text{ mm}$$

$$\text{Bundle cut} = \frac{126.55}{D_b} = 0.244$$

$$R_s = 0.178$$

$$R_w = 0.356$$

$$F_w = 1.04$$



$$A_b = (D_s - D_b) L_b \quad (L_b = \frac{D_s}{4})$$

$$A_b = 1660 \text{ mm}^2$$

$$A_s = 14100 \text{ mm}^2$$

$$N_{CV} = 13$$

Put $N_{CV} = 0$ Let $\alpha = 1.35 \therefore Re > 100$

$$F_b = (0.85) > 0.6$$

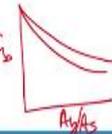
No strip required

$$(C_r = 0.8 \text{ mm})$$

$$N_r = 368 ; N_w = 65 \text{ (round down)}$$

$$A_{nb} = 7234.46 \text{ mm}^2$$

$$D_s = 531.2 \text{ mm}$$

$$\therefore C_s = \frac{1}{16} \text{ in} = 1.6 \text{ mm}$$


So let us start that. So, as far as this F_n correction factor is concerned this will depend on N_{cv} value because Reynolds number in shell side is more than 2,000. So, we have to find out F_n value from the graph which will be based on N_{cv} value. So, first of all let us focus on N_{cv} value and this is the expression for N_{cv} and here it is equal to $D_s - 2H_c / P_t$. So, if you see here we have considered slightly different expression.

Usually, it is $D_b - 2H_b / P_t$, but here I am considering $D_s - 2H_c / P_t$. Now, if you consider the cross sectional view of shell here I am having the baffle. This is the complete cross section of shell and this is basically H_c value. So, D_s is this and H_c is basically the height of cut section of baffle. So, this is D_s , if I am considering baffle cut of another baffle this is further H_c .

So, N_{cv} is what? N_{cv} is basically number of tube rows between two baffle tips. So that I can find as $D_s - 2H_c$ also $/ P_t$. So, both formula will be applicable either you use $D_b - 2H_b / P_t$ or you can use $D_s - 2H_c / P_t$ in both way we can find same N_{cv} value. So, considering these parameters because I already know D_s I also have calculated H_c value which is 132.8.

So, considering all these value we can find out N_{cv} as 12.85 which should be round up and the final value of N_{cv} I am considering as 13. So, you can find out F_n from the graph of F_n and N_{cv} . So, F_n value you can find as 1.01. So, now we have to find out window correction factor and for that I should know R_a . I hope you remember all graphs and steps. Now, if I need to know R_a I have to find out bundle cut.

And bundle cut will be nothing, but H_b / D_b . So, here you have to find out H_b value again. So, H_b should be equal to $D_b / 2 - D_s$ bracket $0.5 - 0.25$ because I am considering 25% baffle cut. So, you can find out H_b value and then bundle cut you can find as 0.244. So, value of bundle cut you can find out depending upon the number of tubes and this is already available in first lecture of week 4.

So, for all these calculation you can please go through that lecture so that you can understand, but I hope if number of tubes are given to you, you can find out bundle diameter. So, here I am having baffle cut as 0.244 and based on that I can find out R_a dash which comes out as 0.178 and so we can find out R_w just twice of R_a dash and after that we can consider graph of F_w .

If you remember we have seen the graph of F_w as well as R_w . So, you can find out F_w value as 1.04 and further we can see bypass correction factor as well as leakage correction factor. Now, if you consider bypass correction factor we have to find out whether sealing strips are required or not and that checking of sealing strip whether it is required or not that method is already explained in the last lecture that is the 5th lecture of 4th week.

So, you can please go through that. If you consider the F_b expression there I am having A_b / A_s and we have N_s / N_{cv} . So, maximum number of sealing strips you can use as 6 because N_{cv} you can have is 13. So A_b and A_s the way we have obtained that in the 5th lecture of week 4 in the same line you can find here also. So, considering all these value if I am putting $N_s = 0$ in F_b depending upon the obtained value of F_b you can decide whether sealing strip is required or not.

So, let us see that calculation A_b you can find out by this where this is L_b and that should be equal to D_s and $- D_b$. So A_b you can find out as this and A_s you have already obtained like this and N_{cv} you have already obtained as 13 and when I am putting $N_s = 0$ in F_b expression. Actually here I am not showing F_b expression because that expression is known to you.

So, I have directly put the value in the expression and here I am showing the values directly. So, if you are using $N_s = 0$ in F_b alpha here is 1.35 as you have Reynolds number more than

100. So, you can find out F b value as 0.853 and you can see that this value is already more than 0.6. It means sealing strips are not required in this case sealing strips are not required. So, here I am considering F b value as 0.853 only.

But here you should find revised F b value from the graph and that graph is between F b and here I am having A b / A s. So, value you can obtain from this graph depending upon the Reynolds number and that value you should consider as final F b value. So, please consider that here I am not considering that value and which is the mistake and I apologize for that because for further calculation I have used this value directly, but that is not the correct method. The correct method I have already explained to you.

So, please consider my apology for this. So, next we will find out leakage correction factor and for that you should find out value of A tb and A sb and in A tb expression you should use C t which is equal to 0.8 mm which is the fixed value in all cases and in A sb you have to find out C s value. So, you consider C t as 0.8 mm.

We have total number of tubes as 368 and N w we can find out as 65 and A tb you can consider as 7234.46 mm square, shell diameter is already given to you. So corresponding to shell diameter you can find out C s value from the table where shell dia and baffle dia are given as per the shell dia range.

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Design of Shell and Tube Heat Exchanger

Solution

$\theta_b = 2.08 \text{ rad}$ = 25% baffle cut

$A_{sb} = \frac{C_s D_s}{2} (2\pi \cdot \theta_b)$

$A_{sb} = 1786.18 \text{ mm}^2$ ✓

$\frac{A_L}{A_s} = 0.639$

$\therefore \beta_L = 0.38$ ✓

$F_L = 1 - \beta_L \left[\frac{(A_{tb} + 2A_{sb})}{A_L} \right]$

$F_L = 0.321$

Hence;
Shell side

$h_s = h_a F_n F_b F_L$

$= 1499.55 \cdot 0.321 \times 0.853 \times 1.04 \times 1.01$

$h_s = 431.29 \text{ W/m}^2\text{ }^\circ\text{C}$

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So, depending upon all these value you can find out A tb and A sb, but for that you also require theta b value. So that theta b value you can obtain as 2.08 and which is corresponding

to 25% baffle cut. So, you can find out A_{sb} value as 1786.18 mm square and then you can find out β_L value from the graph and so the F_L value so F_L value comes over here as 0.321. So, now you have all four correction factors and h_{oc} value. So, you can find out shell side heat transfer coefficient by simply multiplication of all this.

So, simply you have multiplied all values of correction factors with h_{oc} and here you see I have consider F_b as 0.853, but you should consider F_b value from the graph and then you should use that value here. So, final heat transfer coefficient in shell side you can obtain as 431.29 watt per meter square degree Celsius. So, this is the solution of example 2. Now, I am considering another example where we are using Bell's method to find out pressure drop.

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Design of Shell and Tube Heat Exchanger

Example - 3

A fluid with 35000 kg/h is flowing in shell side of a 1-2 shell and tube heat exchanger having following specifications: OD of tube=25 mm, ID of tube=20 mm, shell diameter=1000 mm, number of tubes=600, baffle cut=25%, baffle spacing= $D_s/3$, square pitch arrangement, No. of sealing strips = 4, Density of fluid=550 kg/m³, Viscosity of fluid = 0.09mNs/m².

Compute cross flow zone pressure drop using Bell's method.

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So, here I am having example 3 as a fluid with 35,000 kg per hour is flowing in shell side of 1-2 shell and tube heat exchanger having following specifications where OD of the tube is 25 mm, ID of the tube is 20 mm, shell diameter 1,000 mm, number of tubes are 600, baffle cut 25% and baffle spacing $D_s / 3$. Square pitch arrangement are given and number of sealing strips are 4.

So here we have already fixed the number of sealing strips. So, here we have already fixed the number of sealing strips. Density of the fluid is 550 kg per meter cube and viscosity of the fluid is 0.09 millinewton second per meter square and what we have to compute is cross flow zone pressure drop using Bell's method. So, for this problem we have to find out cross flow zone pressure drop.

And for that you need bypass correction factor as well as leakage correction factor. So, let us start the calculation. So, when you consider the pressure drop in cross flow zone you have to find out ΔP_i and ΔP_i is basically the pressure drop in ideal tube bank. Now, if you recall the expression of ΔP_i it consists $N_{cv} U_s$ and other parameters. So we have to find out U_s that is the shell side velocity at bundle at bundle equator and that will depend on A_s value.

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Design of Shell and Tube Heat Exchanger

Solution

| | | |
|----------------|---------------|------------------------|
| OD of tube | 25 ✓ | mm |
| Shell dia | 1000 ✓ | mm |
| No. of tubes | 600 ✓ | |
| Baffle cut | 25 ✓ | % |
| Baffle spacing | 333.3333333 ✓ | mm $\rightarrow D_s/3$ |
| Mass flow | 35000 ✓ | kg/h |
| Square pitch | 31.25 ✓ | |
| Density | 550 ✓ | kg/m ³ |
| Viscosity | 0.09 ✓ | mN/m ² |
| A_s | 0.066666667 ✓ | m ² |
| U_s | 0.265151515 ✓ | m/s ✓ |
| Re | 40509.25926 | |
| j_f | 0.05 ✓ | |

$$A_s = \frac{(p_t - d_o) D_s l_B}{p_t}$$

So, let us start the solution of this problem if you see here I am having OD of the tube as 25 mm, shell dia 1,000 mm as given, number of tubes are given as 600, baffle cut 25% and baffle spacing which is equal to $D_s / 3$. So 1,000 D_s is given so 333.33 is the baffle spacing, mass flow rate is given to us which is 35,000, square pitch because 25 mm is the OD so you can find out the pitch which is 1.25 into OD.

Density of the fluid is 550, viscosity 0.09 and here we have all expression to calculate A_s and A_s you can find out using this expression and value of A_s you can consider as 0.0667 meter square and in the similar line we can find out shell side velocity at bundle equator which comes out as 0.26515 meter per second. So, considering shell side velocity as well as the property you can find out Reynolds number of shell side.

And it comes out as 40509.25 which is in the turbulent zone and you can consider j_f factor to find out ΔP_i and the j_f factor you can obtain from the same graph which we have used in 5th lecture of 4th week. So that we have already illustrated. So, I am considering value of j_f

directly which comes out as 0.05. So, if you see we have already obtained U s value. Now we have to find out N cv because then only I can calculate delta P i.

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Design of Shell and Tube Heat Exchanger

Solution

| | | | |
|------------|--------|----------|------|
| Bundle dia | n1 | 2.291 | |
| | k1 | 0.156 | |
| | Db | 917.847 | ✓ |
| | Pt' | 31.25 | ✓ mm |
| | Hc | 250 | ✓ mm |
| | Ds-2Hc | 500 | ✓ mm |
| | Ncv | 16 | 16 |
| | Dpi | 123.7374 | Pa |

$$D_b = d_o \left(\frac{N_t}{K_1} \right)^{1/3}$$

$$\Delta P_i = 8 j_f N_{cv} \frac{\rho u_i^2}{2} \left(\frac{\mu}{\mu_w} \right)^{-0.14}$$

| | | | |
|-----|--------|---------|------|
| Fb' | Ns | 4 | |
| | alpha | 4 | ✓ |
| | Ab | 0.02738 | ✓ m2 |
| | Fb | 0.19339 | |
| | Ns/Ncv | 0.2500 | ✓ |
| | Fb' | 0.71273 | |

$$A_b = l_B (D_s - D_b)$$

$$F_b = \exp \left[-\alpha \frac{A_b}{A_s} \left(1 - \left(\frac{2N_s}{N_{cv}} \right)^{1/3} \right) \right]$$



So to find out N cv we have to find out bundle diameter. So, OD of the tube is given and number of tubes are also given which is equal to 600. So, you can consider 1-2 pass and for that k 1 and n 1 value you can obtain corresponding to square pitch and bundle diameter corresponding to 600 tubes you can find using this expression. So, there is nothing new you have already used this expression many times before.

P t dash you can also consider as equal to P t because I am having square pitch now. So, H c value you can consider as 250 D s – 2 H c you can find as 500 and N cv value based on all these you can find as 16 which is basically the total number so you do not have to do anything, you should consider N cv 16 directly. So, once you have N cv value you can find out pressure tube in ideal tube bank using this expression and neglecting viscosity correction factor.

So, based on all these value you can find out pressure drop in ideal tube bank and which is coming out as 123.737 pascal and now we will further calculate bypass correction factor as well as leakage correction factor. So, let us start with bypass correction factor. Now in this problem number of sealing strips are already fixed which is equal to 4 and you have this expression of F b where alpha value is equal to 4 also because it is in the turbulent zone.

So, A_b you can obtain from this because you already know D_s , D_b and L_b . So A_b value you can obtain. Now, considering $N_s = 4$ and N_{cv} as 16 you can find out N_s / N_{cv} as 0.25 and then considering all these values in this expression you can find out F_b dash as 0.7127. So in that way you can find out bypass correction factor for pressure drop calculation. Therefore, we are considering F_b prime.

In the similar line, we should also carry out F_L prime that is leakage correction factor for pressure drop.

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Design of Shell and Tube Heat Exchanger

Solution

| | | |
|------------|----------|-------|
| Hb | 208.9237 | mm |
| Bundle cut | 0.2276 | Hb/Db |
| Ra' | 0.17 | |
| Nw | 102 | 102 |

$$H_b = \frac{D_b}{2} - D_s(0.5 - B_c)$$

$$F'_L = 1 - \beta_L \left[\frac{(A_{tb} + 2A_{sb})}{A_L} \right]$$

$$A_{tb} = \frac{C_t \pi d_o}{2} (N_t - N_w)$$

$$A_{sb} = \frac{C_s D_s}{2} (2\pi - \theta_y)$$

| | | | |
|------------|---------|---------|----------------|
| FL' | Ct | 0.8 | mm |
| | Cs | 4.8 | mm |
| | theta b | 2.1 | |
| | Atb | 0.01565 | m ² |
| | Asb | 0.01004 | |
| | AL | 0.02568 | |
| | AL/As | 0.38527 | |
| | Beta L | 0.48 | |
| | FL' | 0.33238 | |
| Cross flow | DPc | 29.313 | |

| Shell diameter, D_s | Baffle diameter |
|-------------------------------|-----------------------------------|
| Pipe shells | |
| 6 to 25 in. (152 to 635 mm) | $D_s - \frac{1}{16}$ in. (1.6 mm) |
| Plate shells | |
| 6 to 25 in. (152 to 635 mm) | $D_s - \frac{1}{8}$ in. (3.2 mm) |
| 27 to 42 in. (686 to 1067 mm) | $D_s - \frac{3}{16}$ in. (4.8 mm) |

So, to consider that we can use this expression that is F_L prime should be equal to $1 - \beta_L \frac{A_{tb} + 2A_{sb}}{A_L}$ and A_{tb} expression is given over here and A_{sb} expression is given over here which you have already used previously so I am not going into detail of this. Just I am telling that how we can obtain different parameters. So, C_t value is already fixed as 0.8 N_t 600 is already given.

Now you have to find out N_w and for that you have to find out R_a dash because N_w is equal to N_t into R_a dash and for that R_a dash you have to find out bundle cut and that should be equal to H_b / D_b . So, first of all you have to find out H_b using this expression and the value of H_b comes as 208.92. Once you have this value H_b / D_b you can consider which is coming as 0.2276 as bundle cut.

And based on that you can find out R_a dash value as 0.17. Now once you have R_a dash value and total number of tubes you can find out N_w as 102. So, based on that you can find

out A_{tb} value which is coming out as 0.01565 meter square and further I have to find out A_{sb} and for that θ_b value should be seen from the graph along with this C_s is also required.

So, what should be the C_s value? For that we should consider this table where the diameter is 1,000 so which will fall in this range and if I consider C_s value this should be the C_s value. So, in this case C_s value should be 4.8 and considering baffle cut you can find out value of θ_b from the graph. So, baffle cut is 25% and θ_b we can obtain from the graph as 2.1 which we have also seen in last lecture of 4th week.

So you can go through that lecture and considering all these value I can find out A_{sb} as 0.01004 and then we can consider A_L which should be equal to $A_{tb} + A_{sb}$. So, addition of all these will give the value of A_L and after that we can find out ratio of A_L / A_s and based on that I can see the value of β_L from the graph. So that value of β_L comes as 0.48 and based on that I can find out F_L dash value using this equation which is coming at 0.3324.

And accordingly I can find out pressure drop in cross flow zone and that should be equal to ΔP_i into F_b dash into F_L dash. So, in this way you can find out pressure drop in cross flow zone and detail calculations of that I have already explained in the 5th lecture of week 4. However, here some of the steps I have skip because I assume that the method is clear to you and you can simply apply that method and that is the Bell's method.

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References

| | |
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And here I am having some of the references and now as far as this lecture is concerned let me summarize this lecture. In this lecture, we have discussed two different problems. In the first problem, we have calculated shell side heat transfer coefficient using Bell's method and in second example we have discussed pressure drop of cross flow zone using Bell's method. So, I hope the method as well as examples are clear to you and you can apply Bell's method to design shell and tube heat exchanger and that is all for now. Thank you.